

2-1 Conservation laws

- A) Mass
- B) Stress Tensor, Fluxes - momentum
- C) Conservation of Momentum

Batch 73-75, 137-151
 SchL 47-61
 White 61-65

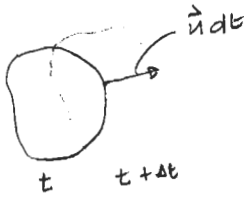
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A) Conservation of Mass

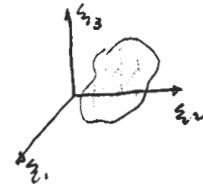
Mass in a C.V : $M(t) = \iiint_V \rho \, dV$

$dV = d\xi_1 \, d\xi_2 \, d\xi_3$

Lagrangian View



assert $\left. \frac{dm}{dt} \right|_{\xi} = 0$



$$\frac{d}{dt} \iiint_V \rho \, dV = \iiint_V \left[\frac{\partial \rho}{\partial t} \Big|_{\xi} \, dV + \rho \frac{\partial V}{\partial t} \Big|_{\xi} \right] = 0$$

$$= \iiint_V \left[\frac{D\rho}{Dt} + \rho \nabla \cdot \vec{u} \right] dV = 0 \quad \text{must hold for any C.V}$$

(last lect $\frac{1}{V} \frac{dV}{dt} = \nabla \cdot \vec{u}$)

Example:



is involved

=> Must have everywhere:

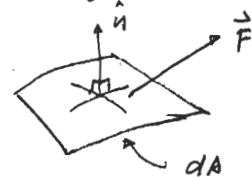
$$\boxed{\frac{D\rho}{Dt} + \rho \nabla \cdot \vec{u} = 0}$$

- conservative form of continuity equations

Eulerian View

(2)

flux of any field quantity $\vec{F}(\vec{x}, t)$ through a C.V.
 $= \vec{F} \cdot \hat{n}$



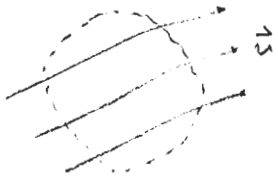
$$d\vec{A} = \hat{n} dA$$

Gauss's Theorem

$$\oint_{C.V.} \vec{F} \cdot \hat{n} dA = \iiint_V \nabla \cdot \vec{F} dV$$

Introduce $\frac{\text{mass flux}}{\text{mass flow}} = \rho \vec{u} \cdot \hat{n}$
 $= \rho \vec{u} \cdot d\vec{A}$

• mass C.V. fixed in space



$$\text{mass in C.V. is } m(t) = \iiint_V \rho dV$$

where $dV = dx_1 dx_2 dx_3$
(local change) +

$$\text{assert } \frac{dm}{dt} = \text{(mass flow in - mass flow out)} = 0$$

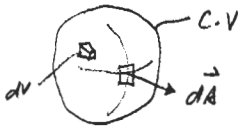
$$\frac{dm}{dt} = \frac{d}{dt} \iiint_V \rho dV + \oint_{C.V.} \rho \vec{u} \cdot \hat{n} dA = 0$$

$$\iiint_V \frac{\partial \rho}{\partial t} dV + \rho \frac{\partial dV}{\partial t} \Big|_x = + \iiint_V \nabla \cdot \rho \vec{u} dV \quad \leftarrow \text{Gauss's Theorem}$$

$$\Rightarrow \iiint_V \left[\frac{\partial \rho}{\partial t} + \nabla \cdot \rho \vec{u} \right] dV = 0 \quad \text{Conservative or Divergence form.}$$

B) Momentum Conservation

Lagrangian View



momentum in C.V. : $\vec{M} = \iiint_V \rho \vec{u} dV$

$dV = d\xi_1 d\xi_2 d\xi_3$

arrives $\frac{d\vec{M}}{dt} = \sum \vec{forces} \text{ on C.V.}$

$\frac{d}{dt} \iiint_V \rho \vec{u} dV = F_{body} + F_{surface}$

Example

$F_{body} = -g \hat{z}$ gravity
 $= \Omega^2 \vec{r}$ centrifugal
 $= 2 \vec{\Omega} \times \vec{u}$ Coriolis

$\frac{d}{dt} \iiint_V \rho \vec{u} dV = \iiint_V \left[\frac{\partial \vec{u}}{\partial t} \rho dV + \vec{u} \frac{\partial (\rho dV)}{\partial t} \right] = F_{body} + F_{surface}$
 $= 0$ conservation of mass.

$\iiint_V \rho \frac{\partial \vec{u}}{\partial t} dV = \iiint_V \rho \vec{f}_{body} dV + \underbrace{\oint \vec{\sigma} \cdot d\vec{A}}_{\iiint_V \nabla \cdot \vec{\sigma} dV \text{ (Gauss)}}$

$\Rightarrow \boxed{\rho \frac{D\vec{u}}{Dt} = \rho \vec{f}_{body} + \nabla \cdot \vec{\sigma}}$ convective form of mass eqn.

$\vec{\sigma}$ is the stress tensor [3x3]

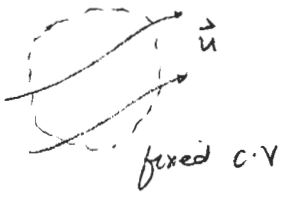
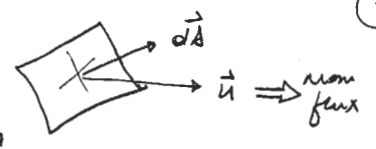
for inviscid flow $\vec{\sigma} = -p \vec{I}$ or $\sigma_{ij} = -p \delta_{ij}$ $\vec{\sigma} = \begin{bmatrix} -p & 0 & 0 \\ 0 & -p & 0 \\ 0 & 0 & -p \end{bmatrix}$

$\Rightarrow \frac{D\vec{u}}{Dt} = \vec{f} - \frac{\nabla p}{\rho}$ - (Euler Equation)

Eulerian View

Introduce $\text{momentum flux} = (\text{mass flux}) \cdot \frac{\text{momentum}}{\text{unit mass}}$

$$= (\rho \vec{u} \cdot \hat{n}) \vec{u}$$



$$M(t) = \iiint_V \rho \vec{u} dV$$

arrives $\frac{dM}{dt} = \sum \text{forces on C-V} + \text{mom. flow in} - \text{mom. flow out}$

$$\frac{d}{dt} \iiint_V \rho \vec{u} dV = \iiint_V \rho \vec{f} dV + \oint_S \vec{\sigma} \cdot d\vec{A} - \oint_{C.V.} (\rho \vec{u} \cdot \hat{n}) \vec{u} dA$$

$$\iiint_V \frac{\partial \rho \vec{u}}{\partial t} dV + \rho \vec{u} \frac{\partial (dV)}{\partial t} \Big|_x = \iiint_V \rho \vec{f} dV + \iiint_V \nabla \cdot \vec{\sigma} dV - \iiint_V \nabla \cdot [(\rho \vec{u}) \vec{u}] dV$$

$$\Rightarrow \frac{\partial \rho \vec{u}}{\partial t} + \nabla \cdot [(\rho \vec{u}) \vec{u}] = \rho \vec{f} + \nabla \cdot \vec{\sigma}$$

divergence or conservative form

where, in cartesian coord system

$$\begin{aligned} \nabla \cdot [(\rho \vec{u}) \vec{u}] &= \frac{\partial}{\partial x_j} (\rho u_j u_i) = \frac{\partial}{\partial x} (\rho u \vec{u}) + \frac{\partial}{\partial y} (\rho v \vec{u}) + \frac{\partial}{\partial z} (\rho w \vec{u}) \\ &= \hat{i} \nabla \cdot (\rho \vec{u} u) + \hat{j} \nabla \cdot (\rho \vec{u} v) + \hat{k} \nabla \cdot (\rho \vec{u} w) \end{aligned}$$

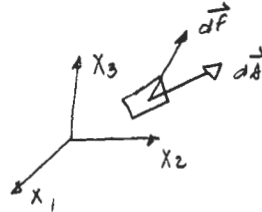
We must still specify $\vec{\sigma}$ to close the system.

Stress Tensor

Surface dF_i acting on an area element dA_i is given by

$$dF_i = \sigma_{ij} dA_j$$

↑
stress tensor defn.



σ_{ij} is like a vector operator which converts an area dA_j into a force dF_j acting on that area.

$$\sigma_{ij} = \frac{\text{force in direction } i}{\text{per unit area in direction } j}$$

σ_{ij} - stress in j direction on plane normal to i

In vector form

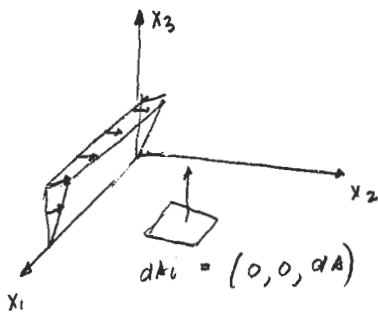
$$\begin{Bmatrix} dF_1 \\ dF_2 \\ dF_3 \end{Bmatrix} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & \sigma_{13} \\ \sigma_{21} & \sigma_{22} & \sigma_{23} \\ \sigma_{31} & \sigma_{32} & \sigma_{33} \end{bmatrix} \begin{Bmatrix} dA_1 \\ dA_2 \\ dA_3 \end{Bmatrix}$$



$$dF_i = \dots$$

Example

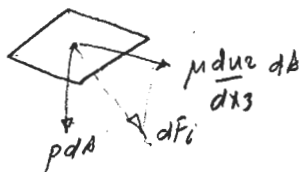
shear flow over a wall



$$u_i = \left(0, \frac{du_2}{dx_3} x_3, 0 \right)$$

$$\sigma_{ij} = \begin{bmatrix} -p & 0 & 0 \\ 0 & -p & \mu \frac{\partial u_2}{\partial x_3} \\ 0 & \mu \frac{\partial u_2}{\partial x_3} & -p \end{bmatrix}$$

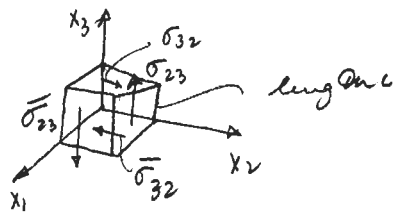
$$\Rightarrow dF_i = \left(0, \mu \frac{\partial u_2}{\partial x_3} dA, -p dA \right)$$



Symmetry is an important property of the stress tensor

(6)

Consider cube with surface stresses.



acceleration

$$a_3 = \frac{\sum F_3}{m} = \frac{l^2 (\sigma_{23} - \bar{\sigma}_{23})}{\rho l^3}$$

$$= \frac{\sigma_{23} - \bar{\sigma}_{23}}{\rho l} \quad \text{as } l \rightarrow 0, a_3 \rightarrow \infty \text{ unless } \underline{\underline{\sigma_{23} = \bar{\sigma}_{23}}}}$$

angular acceleration

$$\alpha_1 = \frac{\sum M_1}{I} \sim \frac{l^3 (\sigma_{23} - \sigma_{32})}{\rho l^4}, \quad \text{as } l \rightarrow 0, \alpha_1 \rightarrow \infty \text{ unless } \underline{\underline{\sigma_{23} = \sigma_{32}}}}$$

Hence σ_{ij} is symmetric

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$$\begin{aligned} * dF_1 &= \sigma_{11} dA_1 + \sigma_{12} dA_2 + \sigma_{13} dA_3 \\ &= \sigma_{11} dx_2 dx_3 + \sigma_{12} dx_1 dx_3 + \sigma_{13} dx_1 dx_2 \end{aligned}$$

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$$\sigma_{11} = \bar{\sigma}_{11} + \frac{\partial \sigma_{11}}{\partial x_1} dx_1$$

$$\therefore dF_{1, net} = \frac{\partial \sigma_{11}}{\partial x_1} dx_1 dx_2 dx_3 + \frac{\partial \sigma_{12}}{\partial x_2} dx_2 dx_1 dx_3 + \frac{\partial \sigma_{13}}{\partial x_3} dx_3 dx_1 dx_2$$

$$\Rightarrow f_1 = \frac{\partial \sigma_{11}}{\partial x_1} + \frac{\partial \sigma_{12}}{\partial x_2} + \frac{\partial \sigma_{13}}{\partial x_3}$$

By for other components given
 $f_{out} = \nabla \cdot \bar{\sigma}_{ij}$