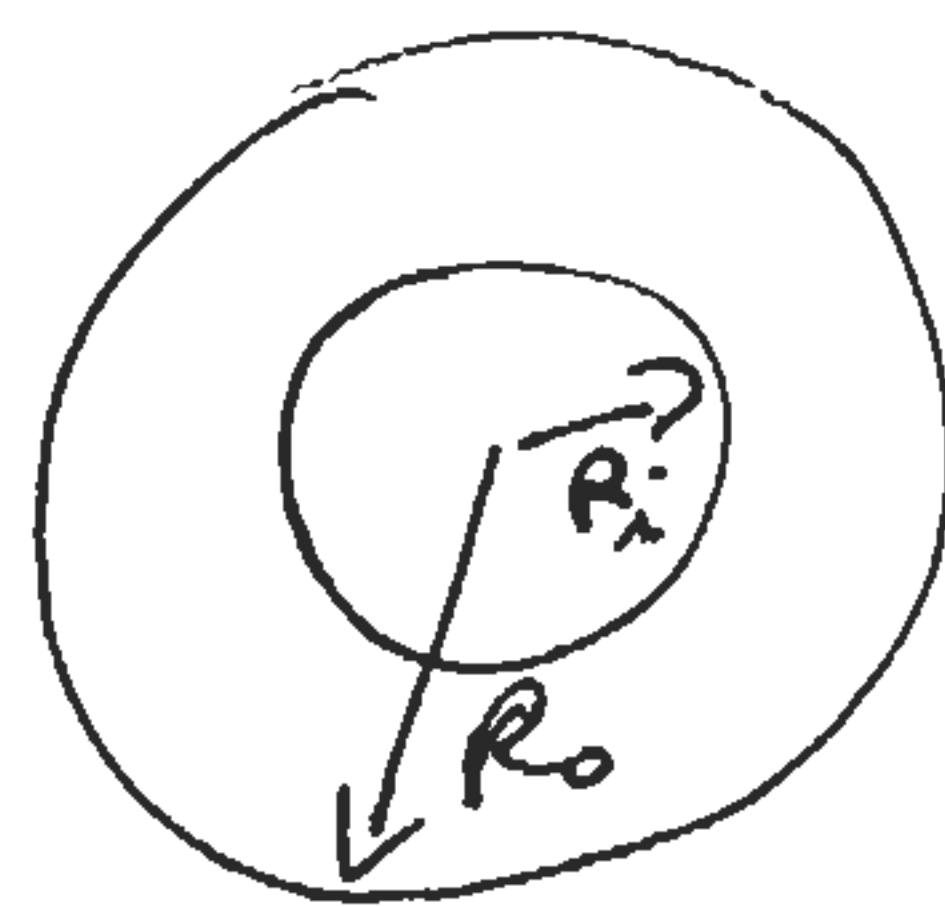


M14

$$E = 70 \text{ GPa}$$



$$R_i = \frac{4}{5} R_o$$

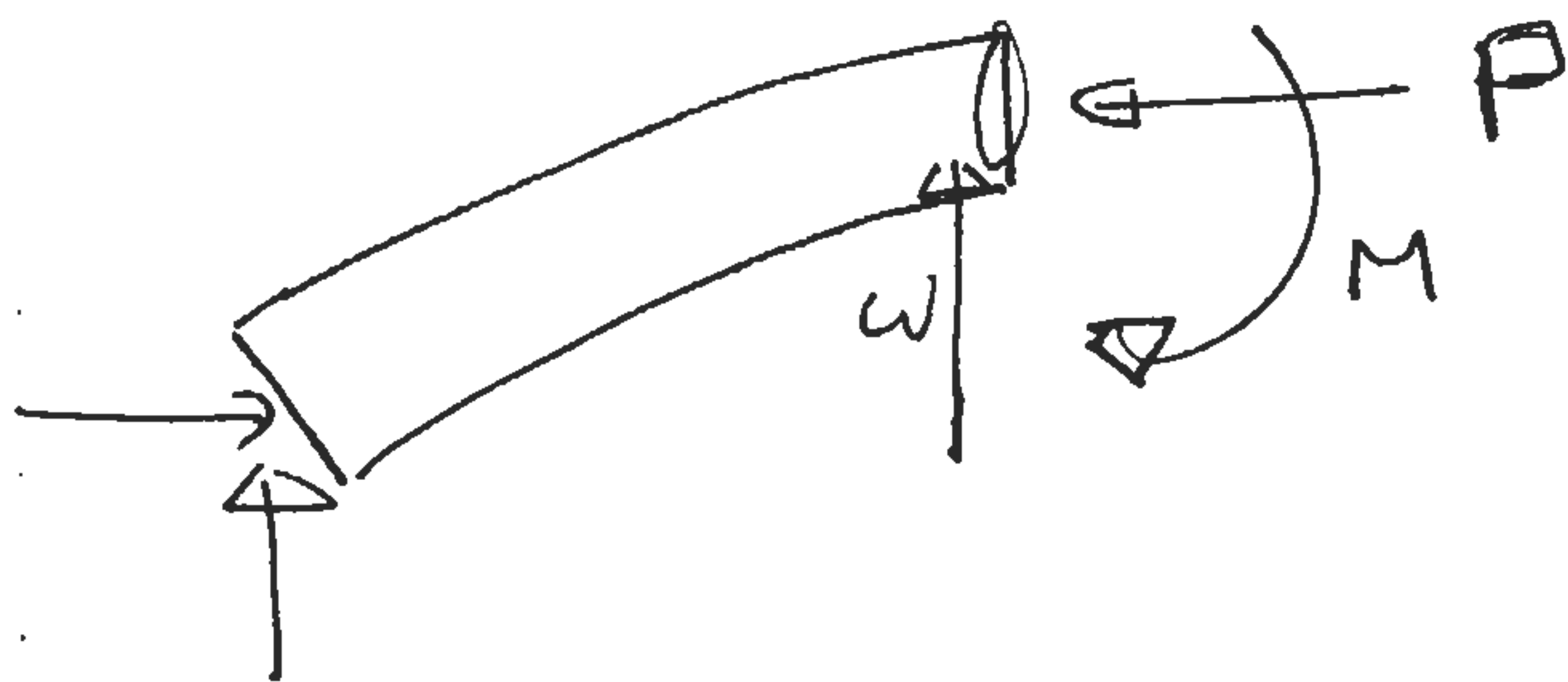
Basic Problem:



which we met in class

$$\therefore w = e \left(\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \sqrt{\frac{P}{EI}} x + \cos \sqrt{\frac{P}{EI}} x - 1 \right)$$

The stress at a given point



$$\sigma = \frac{P}{A} \pm \frac{Mz}{I}$$

axial bending

and $M = EI \frac{d^2 w}{dx^2}$

$$\frac{d^2 w}{dx^2} = -e \left(\frac{P}{EI} \right) \left[\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \sqrt{\frac{P}{EI}} x + \cos \sqrt{\frac{P}{EI}} x \right]$$

Moment will be a maximum at $x = \frac{L}{2}$ since $w = \text{max}$.

$$M = EI \frac{d^2 w}{dx^2}$$

$$M_{max} = -ep \left[\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \left(\sqrt{\frac{P}{EI}} \frac{L}{2} \right) + \cos \left(\sqrt{\frac{P}{EI}} \frac{L}{2} \right) \right]$$

$$\text{let } \sqrt{\frac{P}{EI}} \frac{L}{2} = \theta$$

$$\sin 2\theta = 2 \sin \theta \cos \theta, \quad \cos 2\theta = \cos^2 \theta - \sin^2 \theta$$

$$M_{max} = -ep \left[\frac{1 - (\cos^2 \theta - \sin^2 \theta) \cancel{\sin \theta}}{2 \cancel{\sin \theta} \cos \theta} + \cos \theta \right]$$

$$1 = \cos^2 \theta + \sin^2 \theta$$

$$M_{max} = -Pe \left[\frac{\cancel{\cos^2 \theta} + \sin^2 \theta - \cancel{\cos^2 \theta} + \sin^2 \theta}{2 \cos \theta} + \cos \theta \right]$$

$$= -Pe \left[\frac{\sin^2 \theta \cancel{\cos \theta} + \cos \theta}{\cos \theta} \right]$$

$$= -Pe \left[\frac{\sin^2 \theta + \cos^2 \theta}{\cos \theta} \right] = -Pe \left[\frac{1}{\cos \theta} \right] !$$

$$\therefore M_{max} = -Pe \left[\sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right] \Leftarrow$$

$$\text{max stress} = \frac{P}{A} + \frac{Mz}{I} \quad \text{max for tensile stress}$$

$$A = \pi (R_o^2 - R_i^2) = \pi R_o^2 (1 - \alpha^2)$$

$$\text{let } \frac{r}{s} = \alpha \\ R_i = \alpha R_o$$

$$I = \frac{\pi}{4} (R_o^4 - R_i^4) = \frac{\pi R_o^4}{4} (1 - \alpha^4)$$

$$z = R_o, \quad e = R_o$$

$$\therefore \sigma = \frac{P}{\pi R_0^2 (1-\alpha^2)} + \frac{PR_0}{\pi R_0^4 (1-\alpha^4)} \left[\sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right]$$

$$\sigma = \frac{P}{\pi R_0^2} \left[\frac{1}{(1-\alpha^2)} + \frac{1}{R_0 (1-\alpha^4)} \sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right]$$

Need to iterate to solve. Calculate P_{crit} first = 20 kN.

$$I = \frac{\pi \times (25 \times 10^{-3})^4}{4} \left(1 - \left(\frac{4}{5}\right)^4 \right) = 181.1 \times 10^{-9}$$

$$EI = 12.7 \times 10^3 \quad \frac{L}{2} = 1.25 \text{ m} \quad \frac{1}{25 \times 10^{-3} \left(1 - \left(\frac{4}{5}\right)^4 \right)} = 67.8$$

$$P = 10 \text{ kN. } \sigma = 792 \times 10^6 \text{ Pa ! too high } \frac{1}{1-\alpha^2} = \frac{25}{9}$$

$$P = 1 \text{ kN } \sigma = 38.2 \text{ MPa}$$

$$P = 3 \text{ kN } \sigma = 131 \text{ MPa}$$

$$P = 2 \text{ kN } \sigma = 81.5 \text{ MPa}$$

$$\pi R_0^2 = 1.96 \times 10^{-3}$$

$$\frac{1}{\sqrt{EI}} \frac{L}{2} = 11.10 \times 10^{-3}$$

\therefore max load $\approx 2 \text{ kN}$

$$\text{For perfect column } P_{crit} = \frac{\pi^2 EI}{L^2}$$

$$= \frac{\pi^2 \times 12.7 \times 10^3}{2.5^2} = 20 \times 10^3 \text{ N}$$